

IFAC World Congress - Milano 2011
Workshop on Hierarchical and Distributed MPC:
Algorithms and Applications

Start-Up of Combined Cycle Power Plants

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General context

- Diffusion of Combined Cycle Power Plants (CCPP)
 - Efficiency
 - Lower pollutant emissions
- Production to consumption fit
 - Partial load (ancillary services)
 - Frequent start-up and shut-down
- Flexibility Improvement
 - Reduction of start-up and shut-down time
 - Avoidance of start-up failure
 - Minimization of life-time consumption

CCPP are complex plants with numerous systems and sub-systems

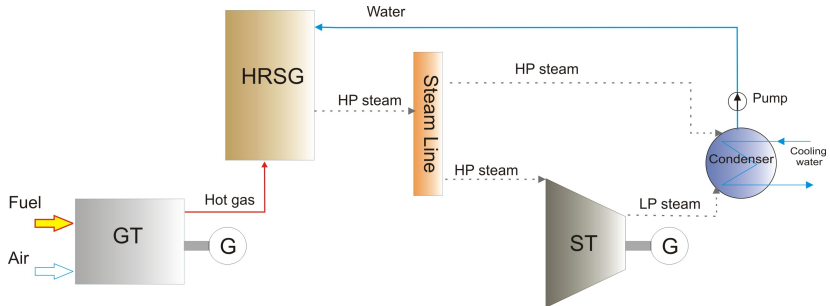


Objectives

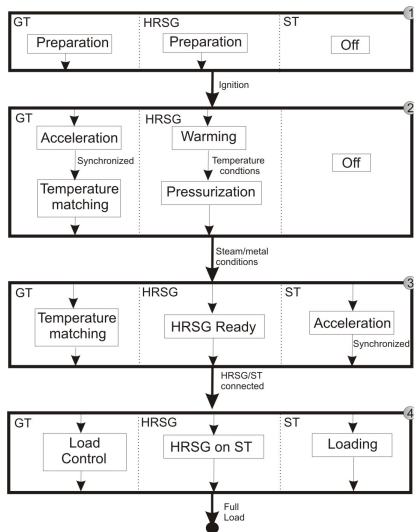
- How can MPC control help to reduce start-up time while saving life-time consumption?
- How can Distribution and Hierarchy help to design and implement control?
- How can design models (Modelica) of the plant be used for operationnal phases?



Schematic view



Start-up procedure

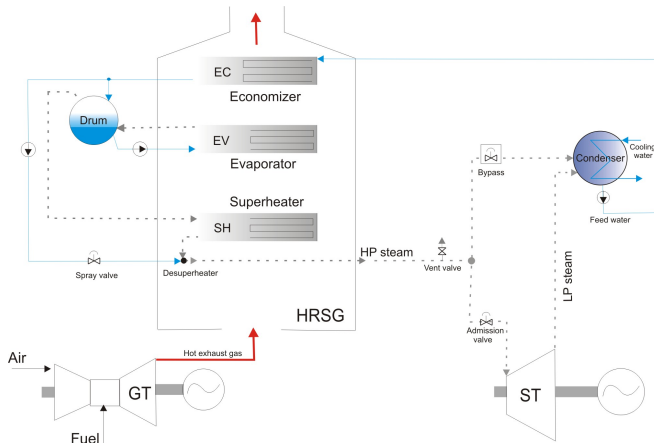


Modelica model Politecnico di Milano (F Casella)

- 1 1 1 CCPP with 3 levels of pressure
- ThermoPower Library
- can be used from low load to high load
- Simplified model:
 - gas turbine
 - low pressure components
- Stress model of critical components
 - high and intermediate pressure superheated steam headers
 - high and intermediate pressure steam turbine rotor



1-1-1 CCPP with 1 level of pressure



Increasing Load Phase

Initial state

- Turbine generators are connected to the grid
- Gas turbine load is around 15%
- Pressure is around 60 e5 Pa
- Steam is admitted in steam turbine (Bypass valve is closed)

Aim: full load

- GT load near 100%
- admission valve: fully open



Local control

- Gas turbine control
- Feed water flow by drum level control loop
- *Steam turbine admission valve by control wrt gas turbine load*
 - when GT load $\leq 50\%$: open-loop control of pressure
 - when GT load $> 50\%$: fully open

Control

- Minimize start-up time
- Constraints on stress level
- Control variables: GTload, *admission valve*



Modelica Model

- ThermoPower library
- Stress model (ASME)
 - Header stress: combination of mechanical and thermal stress
 - Rotor stress: thermal stress
- Complexity
 - ≈ 2400 equations
 - 42 state variables
 - simulation time for increasing load phase: 12.4s (PC with 2 GHz CPU)



1 pressure
level CCPP

ThermoPower Modelica Model



1 pressure
level CCPP

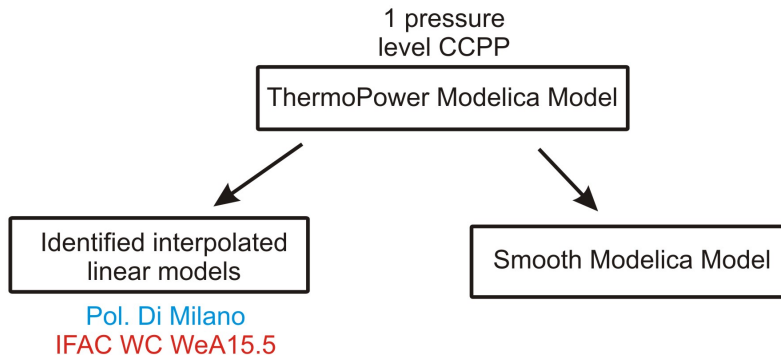
ThermoPower Modelica Model



Identified interpolated
linear models

Pol. Di Milano
IFAC WC WeA15.5





- Aim:
 - elimination of discontinuities
 - *if* clauses
 - piecewise affine functions
 - steam/water tables
 - reduction of simulation time
- Constraint: keep the structure of the ThermoPower model
 - corresponding components
 - new media functions for steam and water



- simplification/specialization of some components (e.g. reverse flow elimination)
- discontinuities approximations

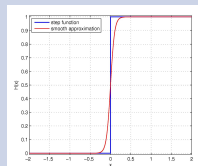


- simplification/specialization of some components (e.g. reverse flow elimination)
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Discontinuity approximation

$$\forall x \in \mathbb{R} \ H(x) = \begin{cases} 0 & x < 0 \\ 1 & x \geq 0 \end{cases}$$

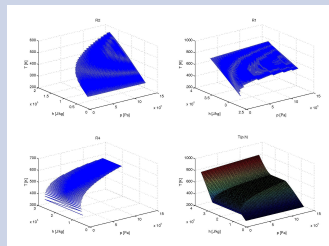
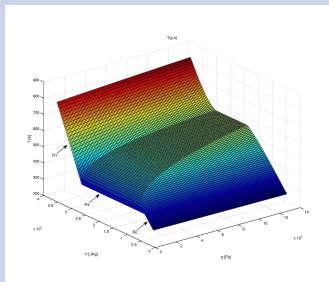
$$\forall x \in \mathbb{R} \ H_k(x) = \frac{1}{1 + e^{-kx}}$$



Steam/Water functions approximation

piecewise polynomial approximations of the Modelica.Media functions

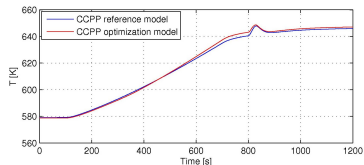
e.g. $T = f(P, h)$



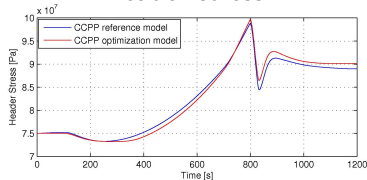
Complexity

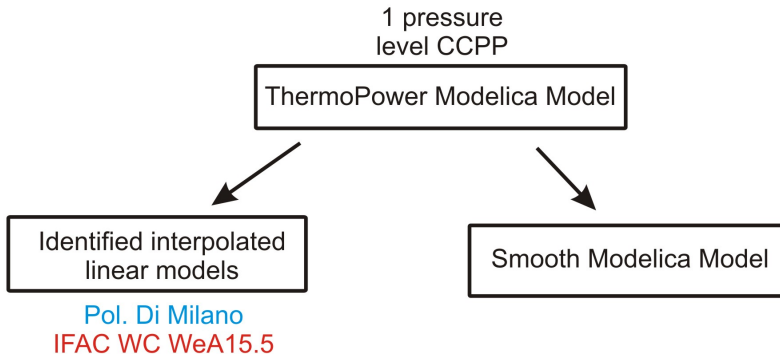
- ≈ 2000 equations
- 42 state variables
- simulation time for increasing load phase: 1.4s (PC with 2 GHz CPU)

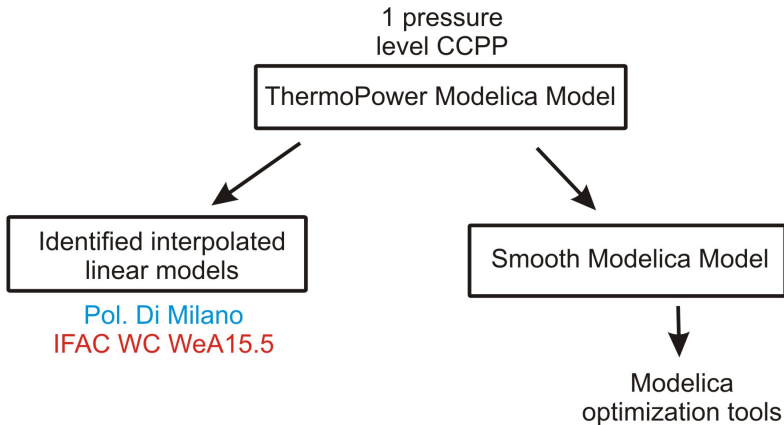
temperature

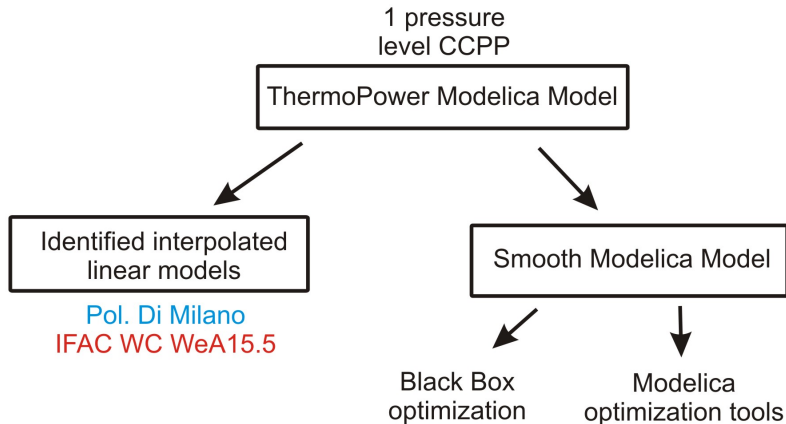


header stress









Profile optimization.

- Choice of parametrized profiles: $L(t) = L_p(t, q)$, e.g.

$$L_{2H}(t, q) = L_m + (L_i - L_m) \frac{t^h}{t^h + k^h} + (L_M - L_i) \frac{t^p}{t^p + r^p}$$

- Optimization problem

$$\min_{t_f, q}(J), \quad J = \int_{t_f}^{t_0} dt$$

subject to the constraints

$$\dot{x} = f(x, L_p(t, q))$$

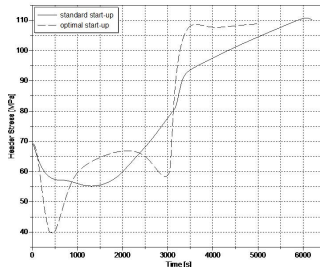
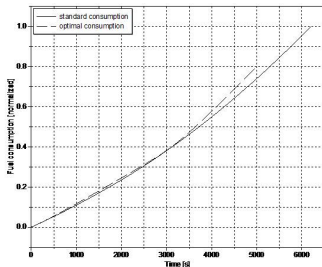
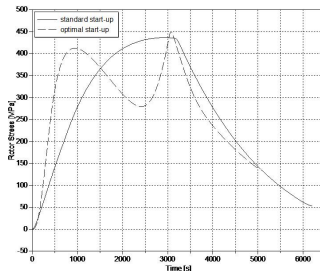
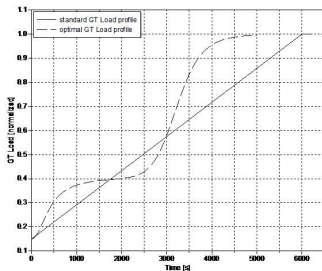
$$L_p(t_f, q) \geq L_M - \epsilon_1$$

$$\|f(x(t_f), L_p(t_f, q))\| \leq \epsilon_2$$

$$h(x(t)) \leq 0$$



Example: $L_{2H}(t, q)$ gains wrt ramp: time 20%; consumption 20%



- Gas turbine load
 - 2 hills functions; start-up time: 4790s (-20%)
 - spline functions (3); start-up time: 4530s (-25%)
- Gas turbine load and steam turbine admission valve
 - spline functions (2); start-up time: 4440s (-26%)



- Control variable: gas turbine load
- Every computation time (T_C)
 - profile computation for the next $N.T_C$
 - Lagrange polynomials (N)
 - minimization of $J = \int_{t_0}^{t_0+N.T_c} \|L_{L_N}(t, q) - L_0(L(t_0))\|^2 dt$



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Start-up time: 3400s (-43%) [$T_C = 60s, N = 5$]



Hierarchical MPC control

- Robustness of control
 - Introduction of variations into the model?
 - Simulation on sets?



Hierarchical MPC control

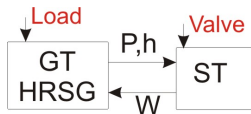
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Distributed control

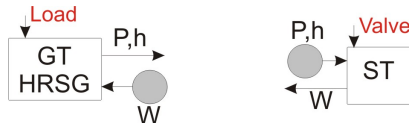
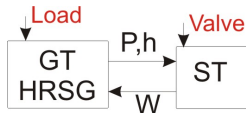
- Gradient based methods?
- Robustness?
- Range of admissible input signals



Example



Example



- Smooth Modelica model for a 1-1-1 1 pressure level CCGP
 - new components / media consistent with ThermoPower
 - systematic design of the optimisation model
- Start-up profile optimization
 - reduction of start-up time
 - importance of profile functions



- Smooth Modelica model for a 1-1-1 1 pressure level CCGP
 - new components / media consistent with ThermoPower
 - systematic design of the optimisation model
- Start-up profile optimization
 - reduction of start-up time
 - importance of profile functions
- Such approach for such plants is still challenging
 - optimization tools / model development
 - simulation tools: admissible state and feasible trajectories
 - distributed approaches: steam interactions

